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Original Publication Citation

Stanley, A. P. J., Ning, A., and Dykes, K., "Optimization of Turbine Design in Wind Farms with Multiple Hub Heights, Using Exact Analytic Gradients and Structural Constraints," *Wind Energy*, Jan. 2019. doi:10.1002/we.2310

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Stanley, Andrew P.J.; Ning, Andrew; and Dykes, Katherine, "Optimization of Turbine Design in Wind Farms with Multiple Hub Heights, Using Exact Analytic Gradients and Structural Constraints" (2019). *All Faculty Publications*. 2986.
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Andrew P.J. Stanley, Andrew Ning, and Katherine Dykes

RESEARCH ARTICLE

Optimization of Turbine Design in Wind Farms with Multiple Hub Heights, Using Exact Analytic Gradients and Structural Constraints

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Abstract

Wind farms are generally designed with turbines of all the same hub height. If wind farms were designed with turbines of different hub heights, wake interference between turbines could be reduced, lowering the cost of energy (COE). This paper demonstrates a method to optimize onshore wind farms with two different hub heights using exact, analytic gradients. Gradient-based optimization with exact gradients scales well with large problems and is preferable in this application over gradient-free methods. Our model consisted of the following: a version of the FLOW Redirection and Induction in Steady-State wake model that accommodated three-dimensional wakes and calculated annual energy production, a wind farm cost model, and a tower structural model, which provided constraints during optimization. Structural constraints were important to keep tower heights realistic, and account for additional mass required from taller towers and higher wind speeds. We optimized several wind farms with tower height, diameter, and shell thickness as coupled design variables. Our results indicate that wind farms with small rotors, low wind shear, and closely spaced turbines can benefit from having two different hub heights. A 9 by 9 grid wind farm with 70-meter rotor diameters and a wind shear exponent of 0.08 realized a 4.9% reduction in COE by using two different tower sizes. If the turbine spacing was reduced to 3 diameters, the reduction in COE decreased further to 11.2%. Allowing for more than two different turbine heights is only slightly more beneficial than two heights and is likely not worth the added complexity.

KEYWORDS:

gradient-based optimization, analytic gradients, structural constraints, different hub heights, FLORIS wake model, tower sizing

NOMENCLATURE

AEP^1	=	annual energy production (GWh)
COE	=	cost of energy (\$/MWh)
α	=	wind shear exponent
x	=	wind turbine x location (m)
y	=	wind turbine y location (m)
z	=	wind turbine hub height (m)
\vec{d}	=	tower diameter vector (m)
\vec{t}	=	tower shell thickness vector (m)
H	=	group hub height (m)

1 | INTRODUCTION

As wind turbines extract energy from the air and convert it to power, an area of reduced wind speed, known as a wake, is formed behind each wind turbine. Because the air in a wake has less momentum, a wind turbine located in a wake cannot extract as much energy and therefore cannot produce as much power. Several solutions have been developed to help remedy this problem, including layout optimization of the wind farm^{1,2,3} and rotor yaw control^{4,5}. It is typical to design wind farms with a single turbine type, which has been designed before any farm layout and control optimization. However, more than one turbine type in a wind farm may help to further decrease negative wake effects. In this paper we will explore the possible benefits of optimizing farm design with different turbine hub heights in decreasing COE. Specifically, we will analyze the benefits of including turbines of two different hub heights in the same onshore wind farm.

Mixed turbine wind farms have been and are currently being studied to increase wind farm productivity. Graf et al. used a genetic algorithm to study layout optimization of a wind farm, while allowing the turbine type to change. They found that the optimal design had two turbine types, and was slightly better than a farm with only one turbine type⁶. Several studies have also already explored the use of different turbine heights in the same wind farm. Chen et al. used a genetic algorithm to optimize a wind farm layout of 25 turbines by changing the position and height of each turbine between two predefined heights. They found that the power increased by as much as 13.5% and the cost per unit of energy produced decreased 0.4%⁷. Hazra et al. used a particle swarm method to optimize a wind farm, in which the turbine height and rotor radius are both design variables. In a 10-turbine wind farm, they found a 12.8% reduction in the cost of power⁸.

More recently, Chen et al. used a greedy algorithm to optimize a wind farm layout and hub heights. Using a particle wake model they found that for wind farms with complex terrain, COE could be decreased by more than 8% when optimizing with two different hub heights rather than just one. They also found marginal benefits using the Jensen wake model for wind farms on flat terrain⁹. Vassel-Be-Hagh and Archer also used a greedy algorithm in which they optimized a wind farm with only hub height as a design variable. They showed a 2% increase in power production when different hub heights are in the same farm. Additionally, Vassel-Be-Hagh and Archer explored situations with two turbines in line with the wind. They showed how the turbine spacing, rotor diameter, and wind shear all affect the power production of the two turbines. Low wind shear, small rotor diameter, and closely spaced turbines produced more power when using different hub heights¹⁰.

Wind farms are large investments, and small improvements of less than 1% can greatly reduce costs and increase profitability over the life of the farm. The results of the studies discussed previously all show significant decreases in the COE of wind farms. All of the mentioned studies indicated that there are farms that can benefit from including turbines with heterogeneous designs—and different hub heights in particular.

Wind farm optimization can be a large problem with many potential design variables and constraints. Optimizing turbine layout introduces two design variables for each turbine. Including tower height introduces another variable per turbine. Other variables can also be added for yaw control, the rotor, tower diameter and thickness, and turbine type. For a wind farm with 25 wind turbines, the problem can easily have thousands of design variables. Gradient-based optimization with analytic gradients is effective for this number of design variables. The number of function calls required to converge using gradient-free optimization scales approximately quadratically with the number of design variables, while analytic gradients scale much better with many

¹AEP in this paper is calculated with directionally averaged wind speeds

design variables¹¹. For small wind farms and few design variables gradient-free methods are acceptable during optimization, as demonstrated in the studies mentioned. However, as the wind farm size increases and the number of design variables increases, gradient-based optimization with analytic gradients becomes necessary⁵.

Differing from previous studies, in our research we used gradient-based optimization with analytic gradients for coupled optimization of many design variables. In this study, we optimize wind farms with up to 567 variables, (hub height, 3 tower diameter variables, and 3 tower thickness variables for 81 different turbines), and our model has been used to optimize over 5000 more coupled variables when we include yaw control. Our work is also unique in that we explore the impact of tower sizing along with the tower height during optimization. Taller towers need a larger diameter and shell thickness because of the associated greater forces and moments, resulting in more mass and higher costs. While having towers with different heights is beneficial in that it will reduce wake interference and produce more energy, the additional tower height of some of the turbines will increase cost. We quantify this relationship between energy production and added mass by optimizing COE, exploring when it is beneficial to have two height groups, and what the tower heights should be for each group. Finally, we examine any additional benefits from having more than two different turbine heights, and even allowing each to vary independently.

The methodology we present in this research specifically applies to onshore wind turbines. We assume that the tower height is the same as the turbine hub height in the loads and cost models. However, by appropriately considering the offshore support structure, and using offshore cost and load models, this study can be applied to offshore wind farms as well. We expect that the results we found for onshore wind farms would be similar to those of an offshore farm, namely that wind farms with mixed hub height wind turbines can result in lower COE.

2 | METHODOLOGY

In this section, we describe the model used to predict the COE of a wind farm. First, the wake model is discussed, which is needed to calculate the wind speed at any point in the wind farm. Next, we discuss the AEP and how it is calculated. We will then describe the structural calculations that were made for each turbine, which were used to constrain the height, diameter, and thickness of the turbines. Finally, we introduce our cost model, and how each of these components is used in optimization.

2.1 | Wake Model

To calculate the effective wind speed at each turbine, we used a modified version of the FLOW Redirection and Induction in Steady-State (FLORIS) wake model originally presented by Gebraad et al⁴. The FLORIS wake model is derived from the Jensen model¹², but rather than use one speed to describe the wind across the wake, three separate zones are defined, each with a different expansion and decay rate. A simple weighted average of the wake overlap is used between zones to define the total effective wind speed at each turbine.

For this study, we assumed constant axial induction of $1/3$ (therefore a constant power and thrust coefficient as well), as done in the original FLORIS wake model formulation. However, we made two modifications to the original FLORIS model. First, we used recent work that has improved FLORIS to provide a smooth response and analytic gradients¹³. These improvements enable solutions to be found for large optimization problems and help to achieve reliable answers. Second, because the original FLORIS wake model was designed to describe wakes in the horizontal plane, it was modified to calculate the effective wind speed at any point in three-dimensional (3D) space. We assume that the wake is axisymmetric, such that any cross section is circular. Additionally, we continue the assumption from the original FLORIS model that the wake center neither ascends nor descends but remains at the same height at which it originated.

A real wake may move in the vertical plane and may not maintain a perfectly circular cross section. To validate the assumptions made, we compared the model results to Simulator fOr Wind Farm Applications (SOWFA). SOWFA is a high-fidelity large eddy simulation tool that was developed at the National Renewable Energy Laboratory (NREL) for wind farm studies^{14,15,16}, and has been used extensively in previous research^{4,17,18}. For more details on this validation, see the work by Stanley et al.¹⁹

2.2 | Annual Energy Production Calculation

The instantaneous power production of a wind farm is dependent on the wind direction, because of the wakes created behind wind turbines. For this reason, AEP is a much better indicator of a productive farm than power. Wind farm AEP takes into account

the power production for all wind speeds and directions as well as the associated frequencies. The wind direction frequency and wind speed data used in this study are from the NoordzeeWind meteorological mast located in the North Sea, measured for one year from July 2005–June 2006²⁰. As seen in the left subfigure of Fig. 1, the direction frequency data is binned into 5° increments with the dominant wind direction being from the southwest. The right subfigure of Fig. 1 shows the directionally averaged wind speeds that were used to calculate AEP in this study.

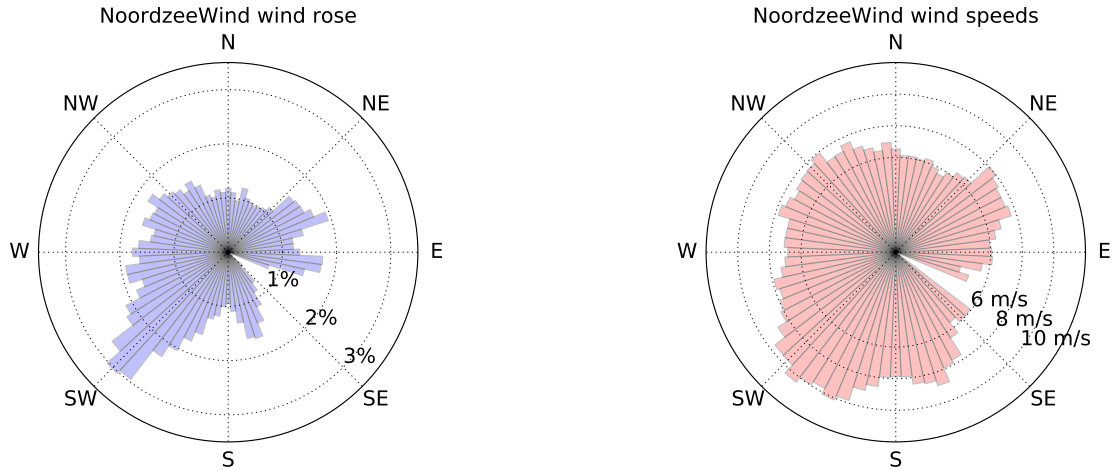


FIGURE 1 On the left is the wind direction frequency distribution from the NoordzeeWind meteorological mast. The data are divided into 72 bins. As can be seen, the predominant wind direction is from the southwest. On the right are the directionally averaged wind speeds, again for the NoordzeeWind meteorological mast.

To account for height differences affecting inflow velocity, we adjusted the wind speed data for wind shear. We used the following power law to estimate the wind speed at different heights

$$U(h) = U_{ref} \left(\frac{h}{h_{ref}} \right)^\alpha \quad (1)$$

where $U(h)$ is the wind speed at any height h , U_{ref} is the reference wind speed (from the Princess Amalia wind data), the reference height h_{ref} is 50 meters, and the shear coefficient α was varied as will be discussed later.

2.3 | Tower Model

Because the tower height was allowed to vary, we included a model to calculate the tower mass and perform structural analysis. The structural analysis was used to constrain the optimization to avoid failure from stress or buckling. For each tower, we calculated values and gradients of von Mises stress, shell buckling, global buckling, tower taper ratio, and the first natural frequency of the entire structure.

The tower mass was calculated from the volume of the tower, and the gradients were calculated analytically. We computed shell buckling as a function of the tower geometry and the stresses at each location, following the method outlined in Eurocode 3²¹. These calculations were made in Fortran 90, and exact gradients were obtained with the Tapenade automatic differentiation tool²². We performed a simple frequency calculation by approximating the tower as a cantilever beam of constant cross section with an end mass. We used the method described by Erturk et al. to calculate the natural frequency²³. Because the turbine tower does not have a constant mass density along the length and the mass from the rotor nacelle assembly is slightly offset at the top, our frequency calculation is slightly more conservative than that predicted by a finite element model by about 10%. For this reason we scaled our frequency calculation by 10% to more closely match reality. We chose this simplified model so that we could find exact gradients, which were obtained using analytic sensitivity equations around the implicit function.

Many calculations in the structural model required the forces and moments caused by the rotor on the top of the tower. In this study, we examine two different rotor sizes. The first rotor had a 126.4 meter diameter, and we used data from the NREL 5-MW

reference turbine for the forces and moments. The second, smaller rotor has a diameter of 70 meters. For the forces and moments of this rotor, we used data from the 1.5–MW NREL WindPACT turbine, which also has a rotor diameter of 70 meters²⁴.

2.4 | Cost Model

AEP is a standard objective in wind farm optimization problems because it is easy to calculate. It is a valid measure when power production is the only dependent variable, but when the tower heights vary, AEP is no longer an appropriate measure. Changing tower height does vary AEP, but it also affects turbine capital cost. To accurately represent this relationship, we evaluated our wind farm by its COE.

COE is defined as:

$$\text{COE} = \frac{\text{FCR}[\text{TCC}(z_i, \vec{d}_i, \vec{t}_i) + \text{BOS}(z_i)] + \text{O\&M}(z_i)}{\text{AEP}(z_i)} \quad (2)$$

where the numerator represents the yearly cost of the wind farm. The FCR is the fixed charge rate, TCC is the turbine capital cost, BOS is the balance-of-station costs, and O&M is the operation and maintenance cost. TCC is a combination of the rotor, nacelle, and tower costs. The rotor and nacelle costs are taken from the turbines after which they are modeled. We approximated the tower cost as a function of tower mass at \$3.08 per kilogram. The BOS costs were calculated using Plant_CostsSE, a model created by NREL to calculate the costs of a wind farm²⁵. In this model, BOS costs are slightly a function of turbine height. Operation and maintenance costs scaled with AEP, and were therefore an indirect function of z_i ²⁶.

2.5 | Optimization

The purpose of this study was to minimize COE of a wind farm. First, we defined the idea of a “height group.” All turbines in a height group had the same tower height, diameter, and shell thickness. We parameterized the tower by specifying the diameter and shell thickness at the bottom, midpoint, and top of the tower and then linearly interpolated diameter and shell thickness at points in between. In this study, there was an equal number of turbines in each height group, or if the number of turbines in the farm was not evenly divisible by the number of groups, some groups had one additional turbine. More height groups are theoretically beneficial because they mean additional freedom in arranging the turbines to decrease wake interference. In Sect. 3.3 we explore the benefits of more than one height group, and how the number of height groups affected the optimized COE.

It may be beneficial to do a discrete optimization in which each turbine can change the height group to which it belongs, but this greatly increases the complexity of the optimization and makes it gradient-free. Discrete variables such as turbine group assignment have no intermediate values, meaning there are no gradients in their optimization. To maintain the gradient-based optimization, we assigned each turbine to one of the height groups before starting the optimization. Once assigned, a turbine could not switch to the other group.

The results of gradient-based optimization, for problems with many local minima, are sensitive to the starting location. As in most optimization problems, there is no guarantee that the solution is the true global solution. However, good results can be achieved with a multiple-start approach, where several different starting points are used for each condition, and the best solution is used. For our study we made hundreds of different random starts. While this is certainly no guarantee that we have found the true global minimum COE value of each condition, the large number of restarts lend confidence that these are good solutions with accurate trends.

Many variables affect the COE of a wind farm, with some more influential than others. In our optimization we considered the following design variables: the tower height of each group (H_j), the tower diameter of each group ($d_{j,k}$), and the tower shell thickness of each group ($t_{j,k}$). Index k refers to the location on the tower ($k=1$ is at the bottom, $k=2$ at the midpoint, $k=3$ at the top).

The tower heights were constrained to be taller than the rotor radius plus a ground clearance, which we set as 10 m. The tower diameter was constrained to be less than 6.3 m for transportation and greater than or equal to 3.87 m at the top, to allow for the connection to the nacelle. Each tower was also structurally constrained by the shell buckling and natural frequency of the tower. The shell buckling constraint was applied to each height group for both the maximum thrust conditions and the survival load, with a safety factor of 1.35 for the loads and 1.1 for buckling resistance. The first natural frequency of the tower was constrained to be greater than the frequency at which the blades rotate and less than the blade passing frequency, with a factor of safety

of 1.1. The diameter-to-thickness ratio was constrained to be greater than 120 at any point, to allow for welding during turbine assembly. The optimization can be expressed as:

$$\begin{aligned}
& \text{minimize} && \text{COE} \\
& \text{w.r.t.} && H_j, d_{j,k}, t_{j,k} \\
& && j = 1, \dots, \text{number of height groups}; k = 1, 2, 3 \\
& \text{subject to} && H_j \geq r_{\text{turbine}} + 10 \text{ m} \\
& && d_{j,k} \leq 6.3 \text{ m} \\
& && d_{j,3} \geq 3.87 \text{ m} \\
& && \frac{3 \Omega}{1.1} \geq f_j \geq 1.1 \Omega \\
& && \text{shell buckling margins: max thrust} \leq 1 \\
& && \text{shell buckling margins: survival load} \leq 1 \\
& && \frac{d_{j,k}}{t_{j,k}} \geq 120
\end{aligned} \tag{3}$$

Note that j is the index describing the height group, and k describes the location on the tower.

The gradients for this optimization were all analytic. We calculated the partial derivatives of each small section of the model and included each part in a framework called OpenMDAO²⁷, which calculates the gradients of the entire system. Each optimization was performed using SNOPT (Sparse Nonlinear OPTimizer), a gradient-based optimization algorithm that works well with problems that have high dimensionality²⁸.

3 | RESULTS

As shown by Vassel-Be-Hagh and Archer, different hub heights are more advantageous in certain conditions¹⁰. In our study, we explored how turbine spacing, wind shear exponent, and rotor diameter affect the optimized COE of wind farms with different hub heights, compared to wind farms with all the same height.

To compare the results, we ran several different optimization cases: a baseline grid wind with no height or layout optimization, a grid with one height group, and a grid with two height groups. These optimizations, their design variables, and how they are described throughout the rest of the paper are shown in Table 1.

TABLE 1 A list of optimizations run in this study. On the left are the names by which they are described throughout the figures and rest of the paper; on the right are the design variables included in each run.

Optimization	Design Variables
baseline	\vec{d}_1, \vec{t}_1
1 group	$\vec{d}_1, \vec{t}_1, H_1$
2 groups	$\vec{d}_{1,2}, \vec{t}_{1,2}, H_{1,2}$

For the optimizations with two height groups, we alternated the placement of each height group to make a checkerboard pattern. Although this pattern seemed logical and produces good results, it may be insightful to consider different height group assignments, such as rows.

When there was no height optimization, the hub heights were set at 90 meters, which is the hub height of the NREL 5-MW reference turbine. For each of the situations listed we optimized two farms, one farm with 25 5-MW wind turbines with 126.4-meter rotor diameters (the large rotor wind farm), and one farm with 81 70-meter rotor diameter turbines (the small rotor wind farm). These rotor diameters were chosen because each is associated with an NREL reference turbine for which we already have rotor load data. The turbines in the small rotor wind farm each had a capacity of 1.543-MW, making the total capacity 125 MW in both the large rotor and small rotor wind farm. The rated wind speed was 11.4 m/s for both the large and small rotors. We

idealized the wind turbines with a cut-in wind speed of zero, meaning the turbines produced power for any non-zero wind speed. This was done because in our AEP calculation, we used directionally averaged wind speeds. If we used a non-zero cut-in speed, certain turbines could produce zero power for a given wind direction if the mean wind speed from that direction is low. In reality, the wind speed in each direction has a distribution and is not constant, meaning at least some power could be produced from a wind direction, even if the mean wind speed is low. Although including a cut-in speed could affect the absolute COE results for some wind farms, we expect the relative differences between different optimization cases would remain similar. However, exploring the effects of different cut-in speeds could be done in future studies. Because we used directionally averaged wind speeds, the cut-out speed was not defined for either rotor, as extreme high wind speeds were never used.

3.1 | Varied Wind Shear

We started by optimizing the turbine design of a grid wind farm through varied wind shear. We expected that two different hub heights would be more beneficial in cases with lower wind shear, because the wind speed at lower heights is almost the same as it is higher up. This means that a lower tower will not experience a significant decrease in wind speed but will be able to reduce wake interference. We varied the wind shear exponent from 0.08 to 0.3, which are realistic extremes for average wind shear (although more extreme values do occur). While varying the wind shear exponent, we explored 2 different square wind farm sizes: the big farm was 2,800 meters by 2,800 meters, and the small farm was 1,680 meters by 1,680 meters. These wind farm sizes and their grid spacing are summarized in Table 2.

TABLE 2 The sizes of the big and small wind farms used in optimization. Sizes are shown absolutely, as well as the grid spacing in rotor diameters.

Wind Farm	Size (m)	Grid Spacing (Small Rotor Diameters)	Grid Spacing (Big Rotor Diameters)
big	2800 by 2800	5	≈ 5.5
small	1680 by 1680	3	≈ 3.3

Figures 2 and 3 show the results of optimizing the small rotor wind farms while varying the wind shear exponent. In each figure, the subfigure in the top left shows the optimized COE as a function of the wind shear exponent. The top right subfigure shows the optimized heights of each of the two height groups, corresponding to the optimization where there are two different height groups. The bottom left subfigure shows the wake loss for each of the optimized wind farms. This wake loss is defined as the percent AEP loss in the wind farm compared to ideal conditions where each turbine in the farm is always exposed to the free stream wind. The bottom right subfigure shows both the ideal AEP and true AEP values for the optimized wind farms. Ideal AEP is the AEP of the wind farm if each turbine were always exposed to free stream wind conditions. The true AEP is calculated in the wind farm, with wake interactions between turbines.

As the shear exponent increases, the benefit of two height groups over one decreases. The COE plot in Fig. 2 shows the results for the big wind farm, where the turbines are spaced the farthest apart. There is a very small benefit to having two height groups for the low shear exponents with the largest COE decrease of 2.2% compared to the optimization with one uniform height group. This benefit decreases with increasing wind shear, and around a shear exponent of 0.22 any benefit is negligible. The top left subfigure of Fig. 3 shows the optimized COE results for the small wind farm. Here, the turbines are very tightly spaced, and there is a significant benefit to different height groups for all wind shear values. The greatest benefit is an 11.2% COE decrease compared to one height group, for a wind shear of 0.08, and 7.1% for the highest wind shear of 0.3. As we expected, two different height groups are more beneficial when there is a low wind shear exponent, which means that the wind speed does not change very quickly with height. These results also indicate that, for a grid layout, two different height groups are much more beneficial in wind farms where the turbines are close together. When the turbines are farther apart, the wakes are much weaker between turbines, meaning that there is not as much benefit from avoiding a wake. When the turbines are very close together the wakes create a very large decrease in wind speed, making it more beneficial to avoid wake interference.

The top right subfigures in Figs. 2 and 3 show the optimized heights of the two different height groups. For the low shear exponent values, there is a large separation between the two heights. This difference decreases as shear increases. In Fig. 2, the

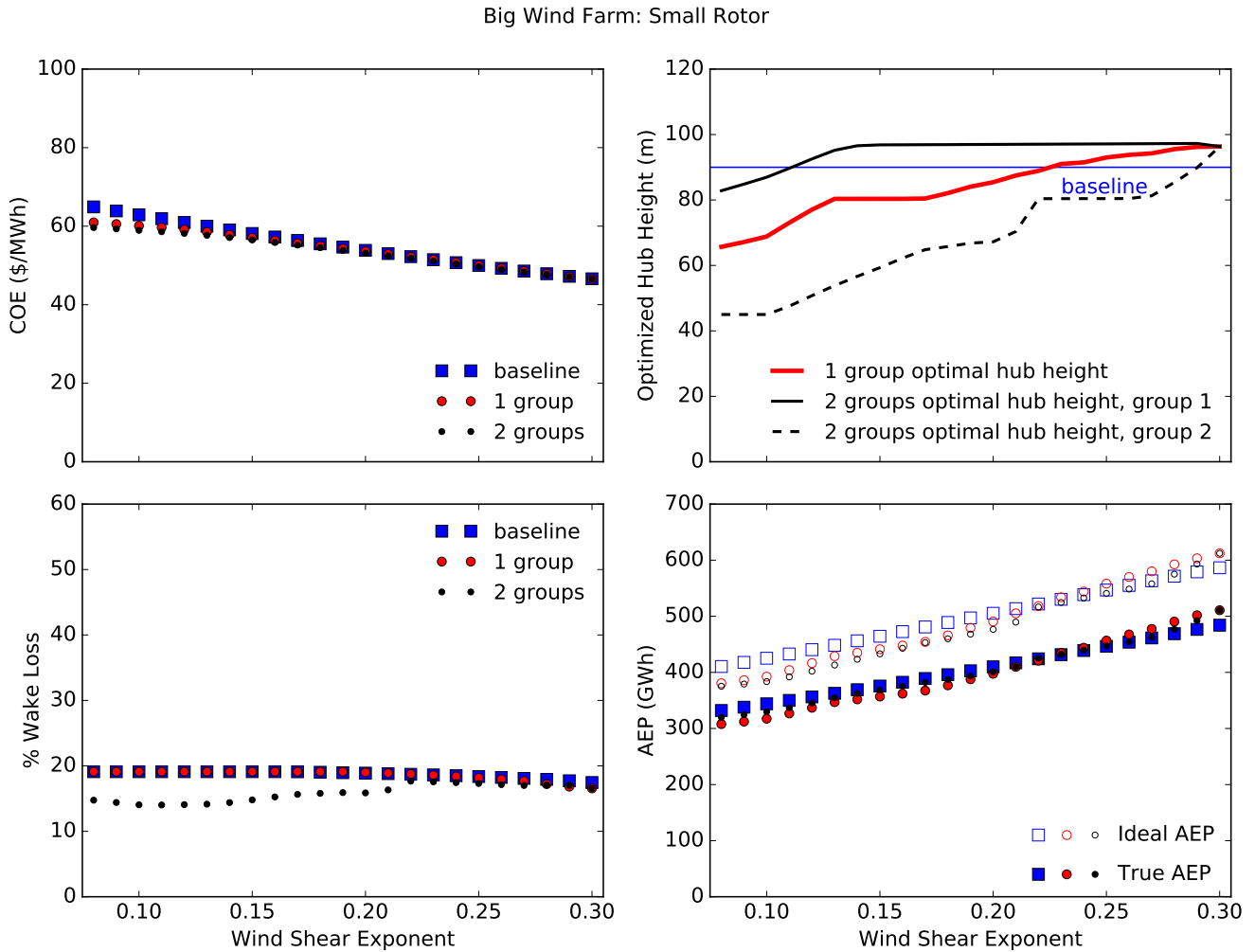


FIGURE 2 Optimization results on a big wind farm, small rotor wind farm while varying the wind shear exponent. The top left figure shows the optimized COE as a function of wind shear exponent. The top right figure shows the optimized hub heights of the optimization runs with two height groups. The bottom left figure shows the percent wake loss for the optimized wind farms. The bottom right figure shows the ideal and true AEP for the optimized wind farms.

highest shear exponent value has no difference between the two height groups, indicating that for this rotor diameter, turbine spacing, and wind shear, there is no benefit to having different height groups. The small wind farm results has a greater difference between the heights of each turbine group for all shear exponents. For the small wind farm, the shorter height group remains at the minimum (45 m) up through a shear exponent of 0.15. The big wind farm has the shorter height group at the minimum for only a few of the lowest shear exponents.

In Figs. 2 and 3, the bottom left subfigure shows the percent AEP loss due to turbine wakes in each optimized wind farm, and the bottom right subfigure shows both the ideal and true AEP values for each of the optimized wind farms. In Fig. 2, for the big wind farm, the wind farm with two groups has lower wake loss than the baseline farm and wind farm with 1 group, up to about a shear exponent of 0.22. After this, there is little to no reduction in wake loss, corresponding to the small separation in optimized hub heights and negligible reduction in optimized COE. Also in Fig. 2, we see that the true wind farm AEP of the wind farms with optimized turbine heights is slightly lower than the baseline case, meaning that the lower optimized COE is a result of lower turbine costs instead of higher AEP. In Fig. 3, for the small wind farm, the wind farms with two different height groups have more than 10% less wake loss than the baseline wind farm and wind farm with one height group. This is from the large separation in the optimized hub heights. In the AEP plot, the ideal AEP of the wind farm with two different heights is lower than the ideal AEP with 1 height group and the baseline case. If these turbines were all exposed to the free stream wind

Small Wind Farm: Small Rotor

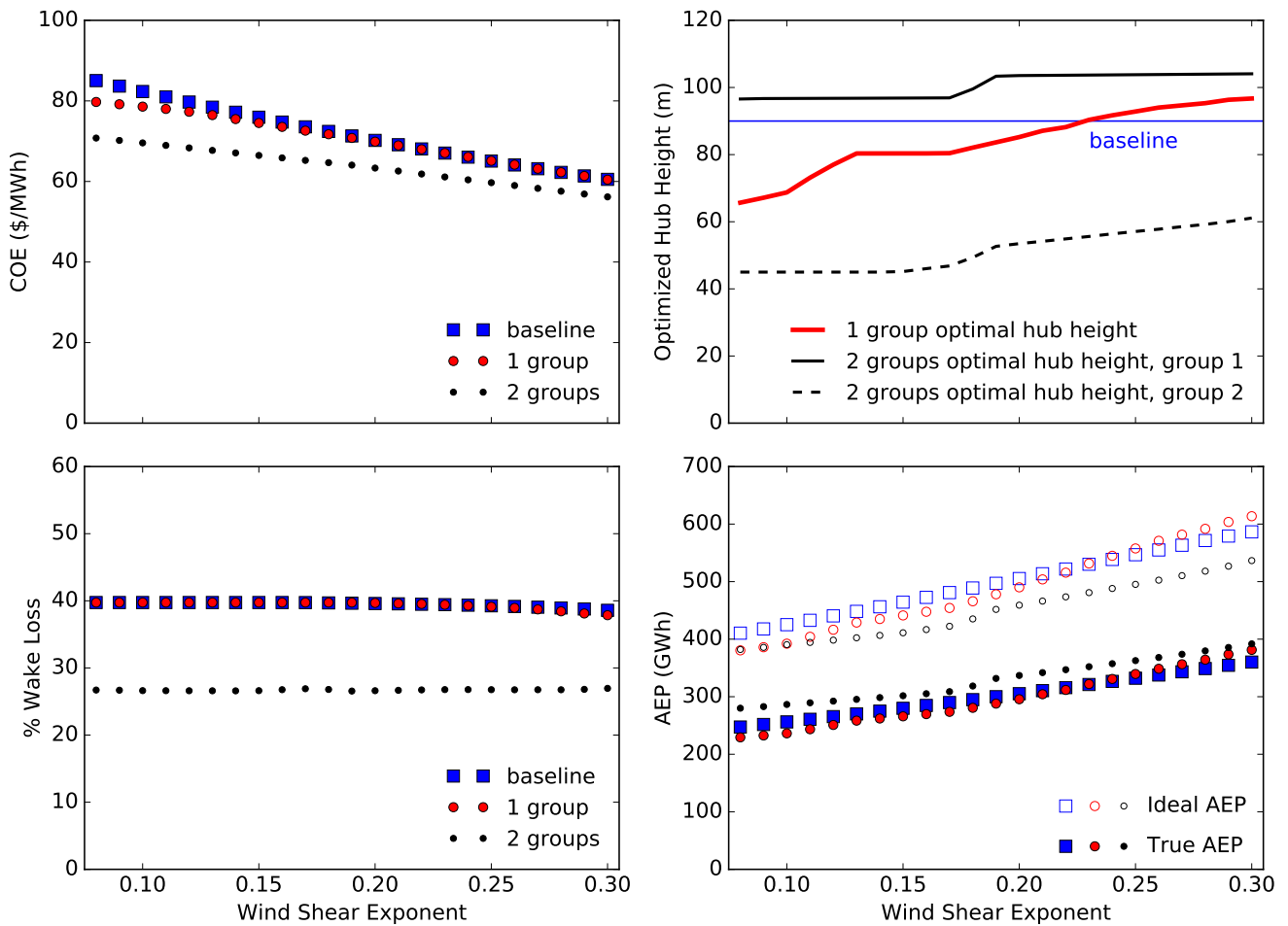


FIGURE 3 Optimization results on a small wind farm, small rotor wind farm while varying the wind shear exponent. The top left figure shows the optimized COE as a function of wind shear exponent. The top right figure shows the optimized hub heights of the optimization runs with two height groups. The bottom left figure shows the percent wake loss for the optimized wind farms. The bottom right figure shows the ideal and true AEP for the optimized wind farms.

conditions, they would perform poorly compared to the other turbine designs. However, in the wind farm, these turbines perform better than the other turbine heights. The lower COE from the wind farms with two different height groups are caused by higher AEP, in addition to potentially lower turbine costs.

Figures 4 and 5 show the results of optimizing a grid, large rotor wind farm while varying the wind shear exponent. Again as in Figs. 2 and 3, the subfigures in the top left show the optimized COE as a function of the wind shear exponent. The top right subfigures show the optimized heights of each of the two height groups, corresponding to the optimization where there are two different height groups. The bottom left subfigures show the wake loss for each of the optimized wind farms. The bottom right subfigures show the ideal AEP and true AEP values for the optimized wind farms.

The trends of the large rotor farm optimizations differ from the small rotor farm. The big farm shows negligible, if any, benefit between the COE for one and two height groups. For the big wind farm, the largest benefit for two height groups is a mere 0.04% at the shear value of 0.12. Even compared to the baseline case, any height optimization results in only a very small COE decrease. The optimized hub heights plot shows that even when two different hub heights are allowed, for almost all shear exponents, it is optimal for the turbines to be the same height. For the big wind farm, the most interesting plot is the AEP, where we see that for the higher shear exponents (above 0.15), optimizing the turbine heights results in a higher AEP. Thus, for a similar COE, more energy can be produced.

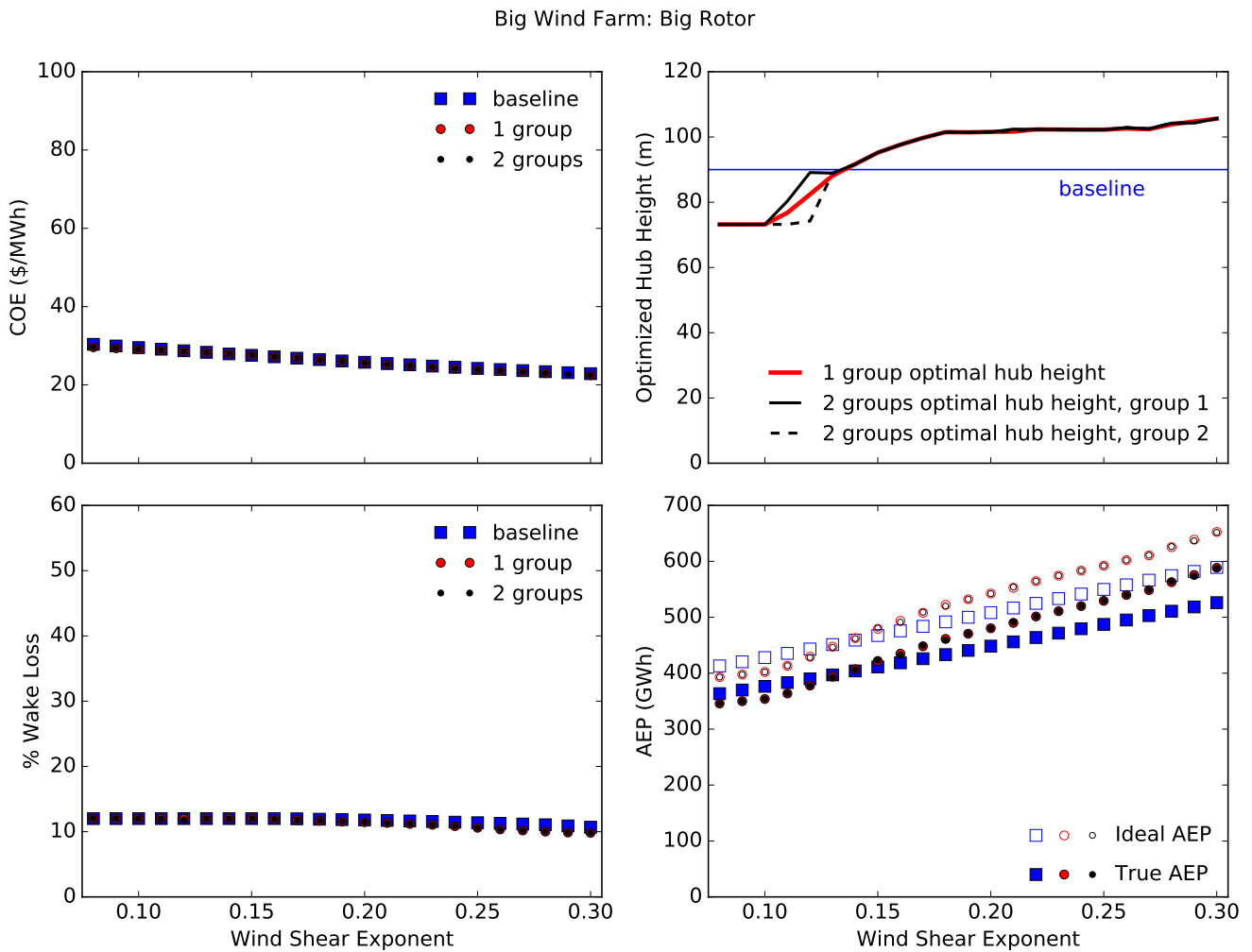


FIGURE 4 Optimization results on a big wind farm, big rotor wind farm while varying the wind shear exponent. The top left figure shows the optimized COE as a function of wind shear exponent. The top right figure shows the optimized hub heights of the optimization runs with two height groups. The bottom left figure shows the percent wake loss for the optimized wind farms. The bottom right figure shows the ideal and true AEP for the optimized wind farms.

For the small wind farm results in Fig. 5, there is a small but noticeable benefit between one and two height groups in the shear exponent range 0.12-0.18. The largest benefit is 1.2% at the 0.14 shear exponent. For this same range of shear exponents, there is a noticeable difference in the optimized hub heights of each group, as well as a noticeable decrease in the percent wake loss of the wind farms with two height groups. Again, as with the big wind farm, the wind farms with optimized hub heights result in a higher AEP for shear exponents above 0.15.

The benefits from two different height groups are about ten times greater for the small rotor wind farm than the large rotor wind farm. The major reason for this difference is the relative size of the rotor compared to the height of the turbine. For the small wind farms with a shear exponent of 0.15, the height groups are greatly separated for both the big and small wind farms (see the top right subfigures of Figs. 3 and 5). The small rotor wind farm has a height difference of just over 50 m (Fig. 3), and the big rotor wind farm has a height difference of about 33 m. Figure 6 shows what these height differences actually look like for the different rotor diameters. For the small rotor case, the 50 m difference in hub heights results in very little overlap between the rotors of the different height groups. Even when directly in line with the wind direction, the wakes from one height group will not greatly affect the turbines in the other group. Compare this to the large rotor farm. Even at maximum separation, there is a large overlap between the rotors of each height group. The wakes of each group will always greatly impact the power production of the other.

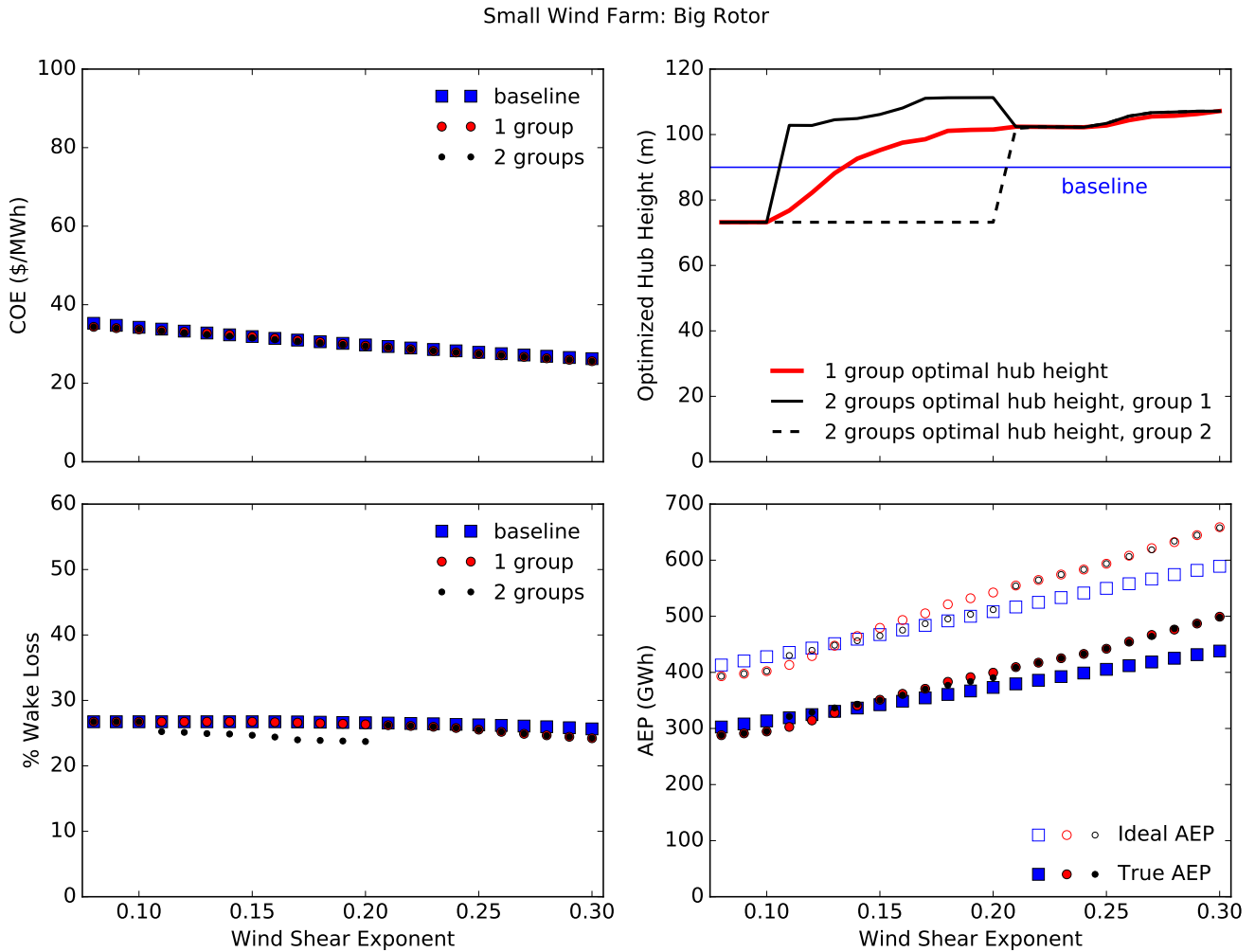


FIGURE 5 Optimization results on a small wind farm, big rotor wind farm while varying the wind shear exponent. The top left figure shows the optimized COE as a function of wind shear exponent. The top right figure shows the optimized hub heights of the optimization runs with two height groups. The bottom left figure shows the percent wake loss for the optimized wind farms. The bottom right figure shows the ideal and true AEP for the optimized wind farms.

3.2 | Varied Turbine Spacing

Next, we explored how different height groups benefit farms with different turbine spacing. We investigated this with a metric called turbine density. Turbine density is a measure of the average turbine spacing in the wind farm and is defined as the area of all of the rotor disks relative to the total farm area:

$$\text{Turbine Density} = \frac{\pi R^2 N}{A} \quad (4)$$

where R is the rotor radius, N is the number of turbines, and A is the area of the wind farm. We varied the turbine density by changing the total area of the wind farm, while keeping the number of turbines and their rotor diameter the same. Although in this study only grid wind farms are considered, turbine density (instead of grid spacing, which is a more traditional turbine spacing metric) is considered in preparation for future studies which will couple turbine design and layout optimization. In such studies when the layout is not necessarily a grid, average grid spacing will no longer be intuitive. For this portion of the study, we ran density sweeps with two shear exponent values: 0.08 and 0.25.

Figures 7 and 8 show the results of our optimization runs while varying the turbine density for a grid wind farm with a small rotor and a shear exponent of 0.08 and 0.25, respectively. Each of the subfigures shows the same information as Figs. 2–5 with

Optimized Wind Turbine Heights

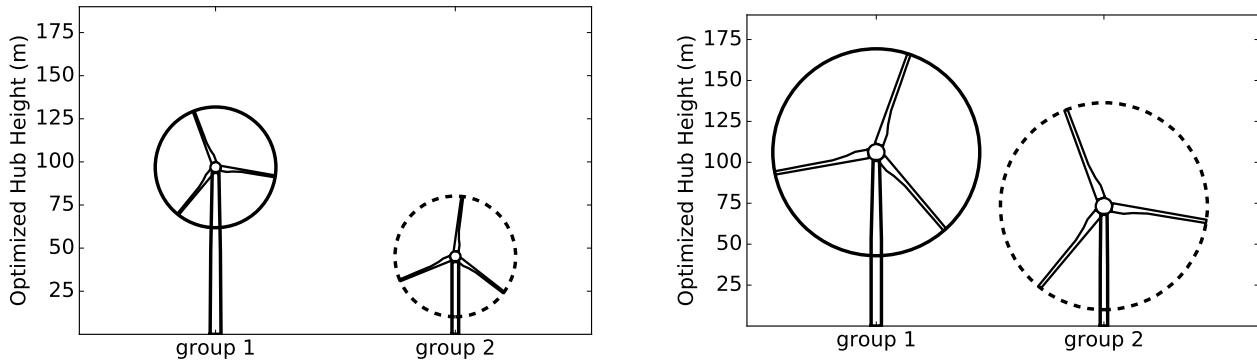


FIGURE 6 The two optimized height groups for the small wind farm at a shear exponent of 0.15. On the left is one turbine from each height group in the small rotor wind farm; on the right is one turbine from each height group of the big rotor wind farm.

the varied wind shear. The top left shows COE, the top right shows optimized hub heights, the bottom left shows wake loss, and the bottom right shows ideal and true AEP.

In Figs. 7 and 8, it is apparent that at high turbine densities, two height groups have a much lower COE than one height group. As the turbine density decreases, this benefit also decreases. Also, the lower shear exponent of 0.08 in Fig. 7 has greater COE reduction with two height groups than a shear exponent of 0.25 in Fig. 8. In Fig. 7, at a shear exponent of 0.08 and at the highest turbine density, there is a 23.3% decrease in COE from one height group to two. In Fig. 8, at a shear exponent of 0.25 and the highest turbine density, the COE decrease is 20.6%. Also notice the COE of one height group is identical to the baseline grid at the shear exponent of 0.25. This occurs because the optimized height of the one height group approaches the baseline height of 90 meters.

The top right subfigures of Figs. 7 and 8 show the optimized hub heights of the two height groups. For both shear exponents and for almost all turbine density values, there is a large difference between the two height groups (over 50 meters). Only below a turbine density of 0.1 do the hub heights get significantly closer together, for both shear values.

The bottom subfigures of Figs. 7 and 8 show the wake loss in each optimized wind farm and the ideal and true AEP values for each optimized wind farm. For each shear exponent, these figures are very similar. With two height groups, there is much lower wake loss than when there is only one height group. This is true across all the turbine densities except for the very lowest, when the turbines are very far apart (close to 10 rotor diameters). Also, as in Fig. 3, across almost all turbine densities the ideal AEP is lower with two height groups, but the true AEP in the wind farm is higher. The only exception is in Fig. 8 for turbine densities below 0.05. Again, this means that for the small rotor diameter, wind farms with different hub heights can produce a higher AEP, and have a lower COE.

Wind farms with high turbine density have turbines that are very close together, meaning that there is significant wake interference. For these farms, there is a benefit to having different hub heights, to avoid wakes between turbines. At low turbine density the turbines are farther apart, meaning that the wakes are much weaker and the wind speed is higher. In these situations, having two height groups is not as beneficial.

Figures 9 and 10 show the results of our optimization runs while varying both the turbine density for a grid wind farm with a large rotor and shear exponents of 0.08 and 0.25, respectively. Again, the top left of each figure shows COE, the top right shows optimized hub heights, the bottom left shows wake loss, and the bottom right shows ideal and true AEP.

These results behaved differently than originally expected. While the benefit of two height groups over one does increase with increasing turbine density for the large rotor wind farm, it is not fully realized at the lowest shear exponent, 0.08. For both shear exponents, 0.08 and 0.25, the benefit of having two height groups over one at the highest turbine density is 0.7% and 2.5%, respectively. At a shear exponent of 0.08 in Fig. 9, there is a small benefit to having two height groups only at the very highest turbine densities (0.225–0.25). For a shear value of 0.25 in Fig. 10, there is a significant benefit to two height groups, but only down to a turbine density of around 0.175. The reason for this behavior is explained in the top right subfigures of Figs.

0.08 Shear Exponent: Small Rotor

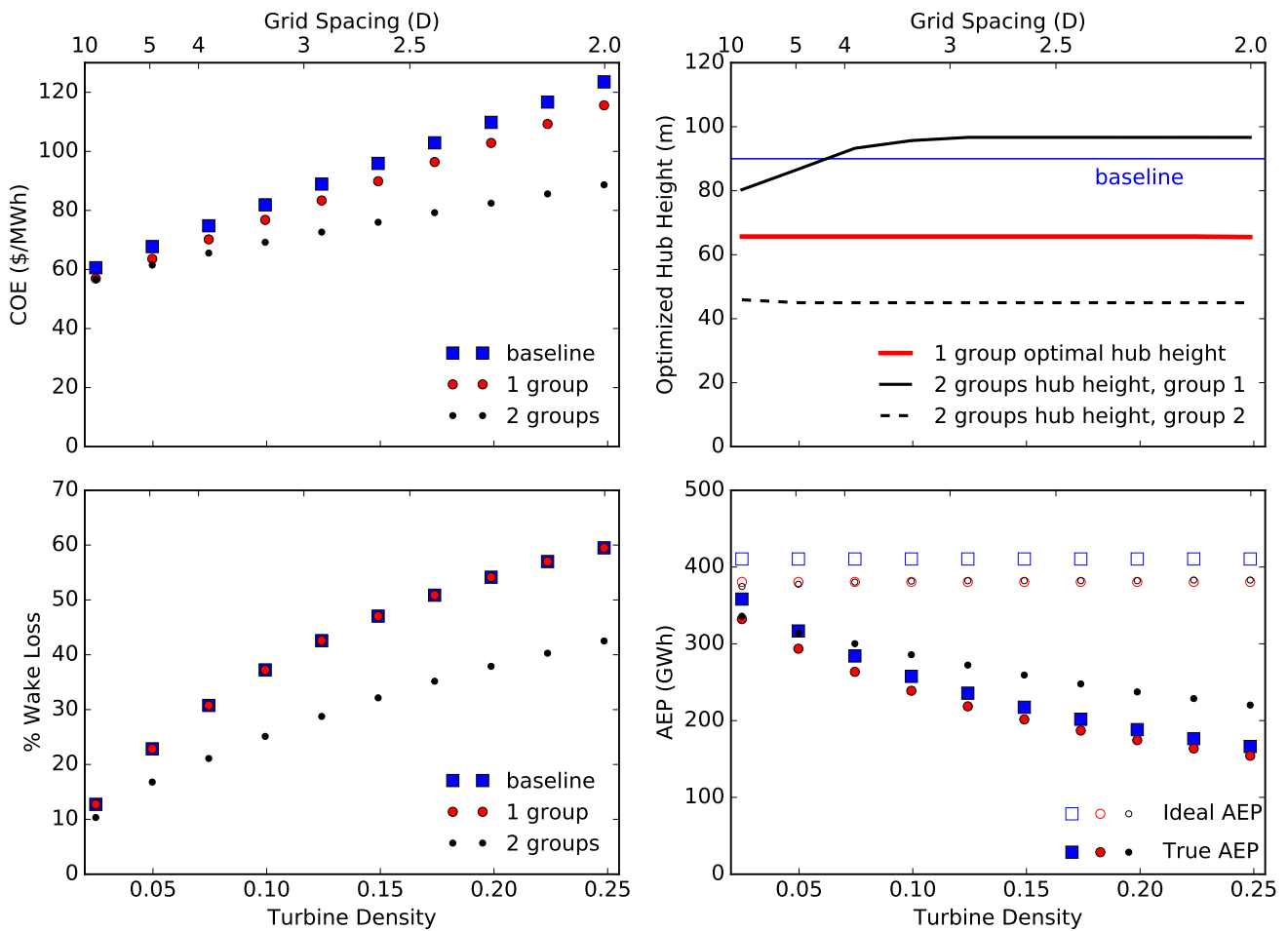


FIGURE 7 Optimization results of a small rotor wind farm while varying the turbine density with a wind shear exponent of 0.08. The top left figure shows the optimized COE as a function of wind shear exponent. The top right figure shows the optimized hub heights of the optimization runs with two height groups. The bottom left figure shows the percent wake loss for the optimized wind farms. The bottom right figure shows the ideal and true AEP for the optimized wind farms.

9 and 10, which show the optimized turbine heights. In Fig. 9, the tower heights of both groups are at the height minimum (73.2 meters) for the majority of turbine densities. At this shear exponent of 0.08, the higher wind speed and power production gained from a higher tower is not worth the extra cost. In fact, it would result in a lower COE to have a shorter tower, but this value is constrained at 73.2 meters to allow sufficient clearance between the ground and the blades. Now contrast this behavior with Fig. 10. Through a turbine density of 0.125 there is no separation between the height groups. However, below this turbine density the optimized height does not go to the minimum but remains much higher, over 100 meters. At this shear exponent and turbine densities, it is not worth the decrease in wind speed to have one shorter height group.

When turbine heights are optimized coupled with turbine layout or yaw there is a clear benefit from the additional design variables; however, the trends for the benefit of two height groups remain the same. Every point in the COE portions of Figs. 2–5 and 6–10 shifts down approximately equally with the other points in the same figure, keeping the benefit of two height groups over one the same for each case. When layout and yaw control are also included as design variables, two height groups are still more beneficial in wind farms with low wind shear, high turbine densities, and small rotor diameters.

0.25 Shear Exponent: Small Rotor

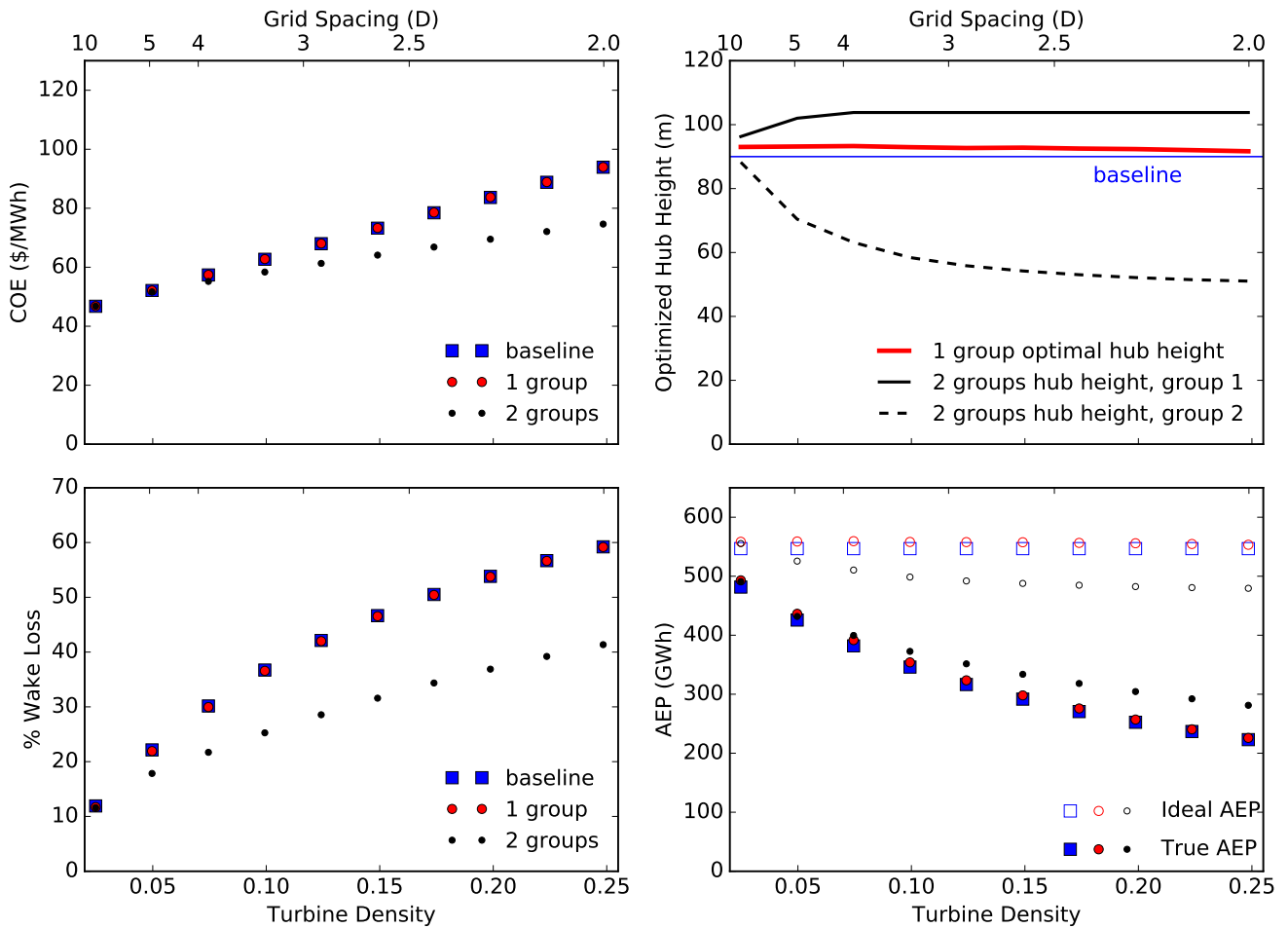


FIGURE 8 Optimization results of a small rotor wind farm while varying the turbine density with a wind shear exponent of 0.25. The top left figure shows the optimized COE as a function of wind shear exponent. The top right figure shows the optimized hub heights of the optimization runs with two height groups. The bottom left figure shows the percent wake loss for the optimized wind farms. The bottom right figure shows the ideal and true AEP for the optimized wind farms.

3.3 | Effect of the Number of Height Groups

Up to this point, the highest number of height groups in a farm has been two. However, if there is a large decrease in COE from one height group to two, it is logical to think there could be additional benefits to adding more height groups, or even letting each turbine vary its height individually. This would add complexity to the design and manufacture of a wind farm, but large COE reduction could be worth the additional complexity.

To study the effects of additional height groups, we optimized the small wind farm with a grid layout, and the wind shear exponent at 0.15. We then varied the number of height groups in the optimization from 1 up to the number of turbines in the farm. We randomized which turbines were assigned to each height group, but found that the lowest COE occurred when height groups are arranged in rows such that the optimizer could stagger the turbine heights in line with the dominant wind direction. Figure 11 shows the results for optimizing the wind farms with different numbers of height groups. On the left are the results for the small rotor wind farm, while on the right are the results for the large rotor farm.

There is a large COE decrease when optimizing with two height groups as compared to just one; however, adding more height groups after that is not tremendously beneficial. In Fig. 11 on the left, for the small rotor farm, there is a 13.5% COE decrease from one height group to two and only a 1.0% decrease from 2 height groups to 81, where each turbine can vary independently.

0.08 Shear Exponent: Big Rotor

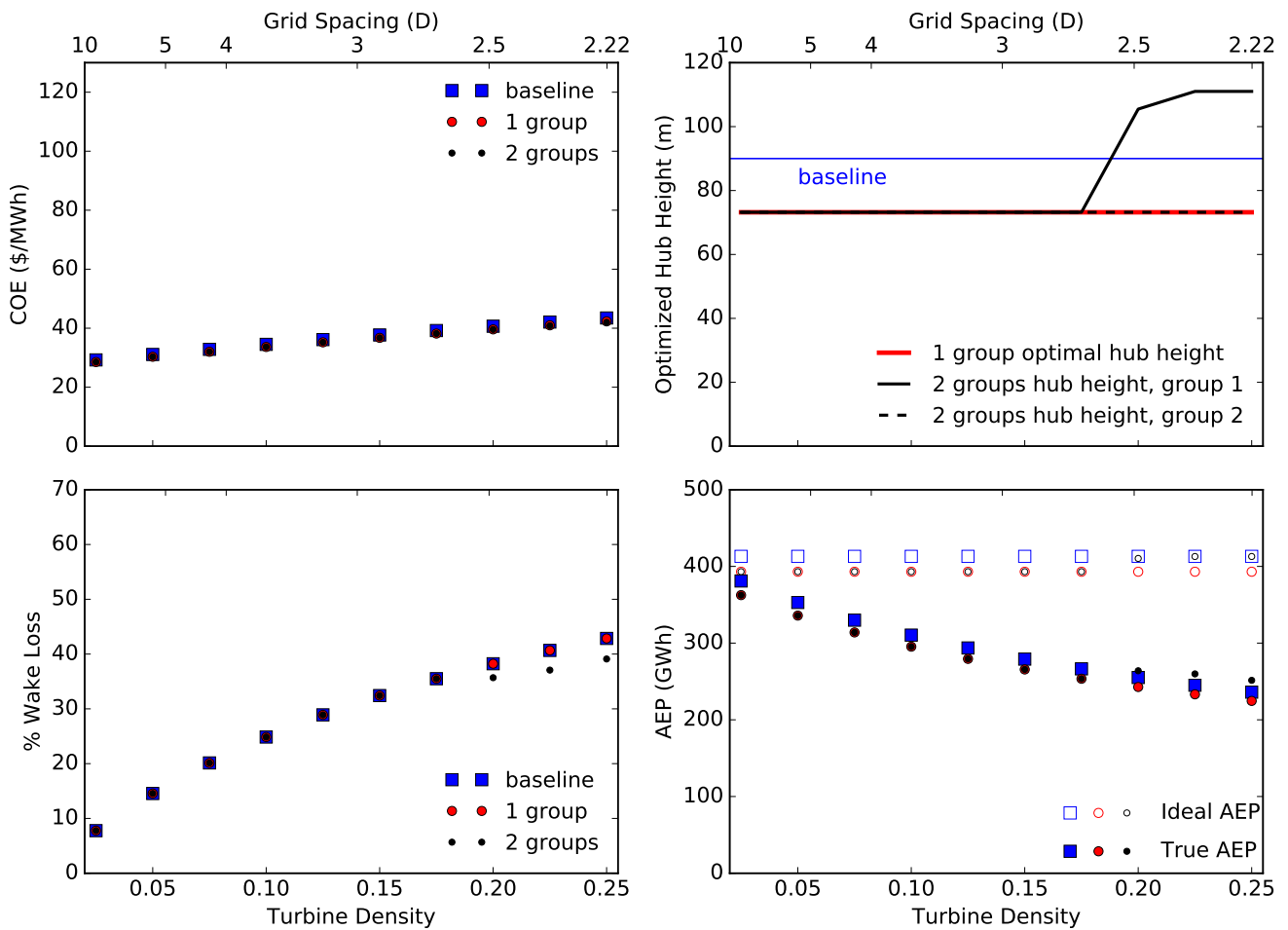


FIGURE 9 Optimization results of a big rotor wind farm while varying the turbine density with a wind shear exponent of 0.08. The top left figure shows the optimized COE as a function of wind shear exponent. The top right figure shows the optimized hub heights of the optimization runs with two height groups. The bottom left figure shows the percent wake loss for the optimized wind farms. The bottom right figure shows the ideal and true AEP for the optimized wind farms.

On the right, for the large rotor farm, there is a 1.6% decrease in COE from one height group to two and a 0.07% decrease from 2 height groups to 25. From one height group to two, the large COE reduction appears to be worth the small added complexity in wind farm design and manufacturing. However, adding more height groups or letting each turbine be optimized individually does not seem to be worth it.

4 | CONCLUSIONS

This research presented a method to optimize a wind farm with different hub heights using exact analytic gradients and structural constraints. We discuss how to account for wake interference between wind turbines of different hub heights, wind shear, structural calculations to constrain the tower during optimization, and the cost of the wind farm. This paper is unique compared to other research in this area in that we use structural constraints, and sized the entire tower. We explored how the increased AEP from different hub heights interacts with the additional costs from taller towers by optimizing COE. It is also unique in that we used a gradient-based approach, which allowed us to optimize much larger farms with more design variables than are

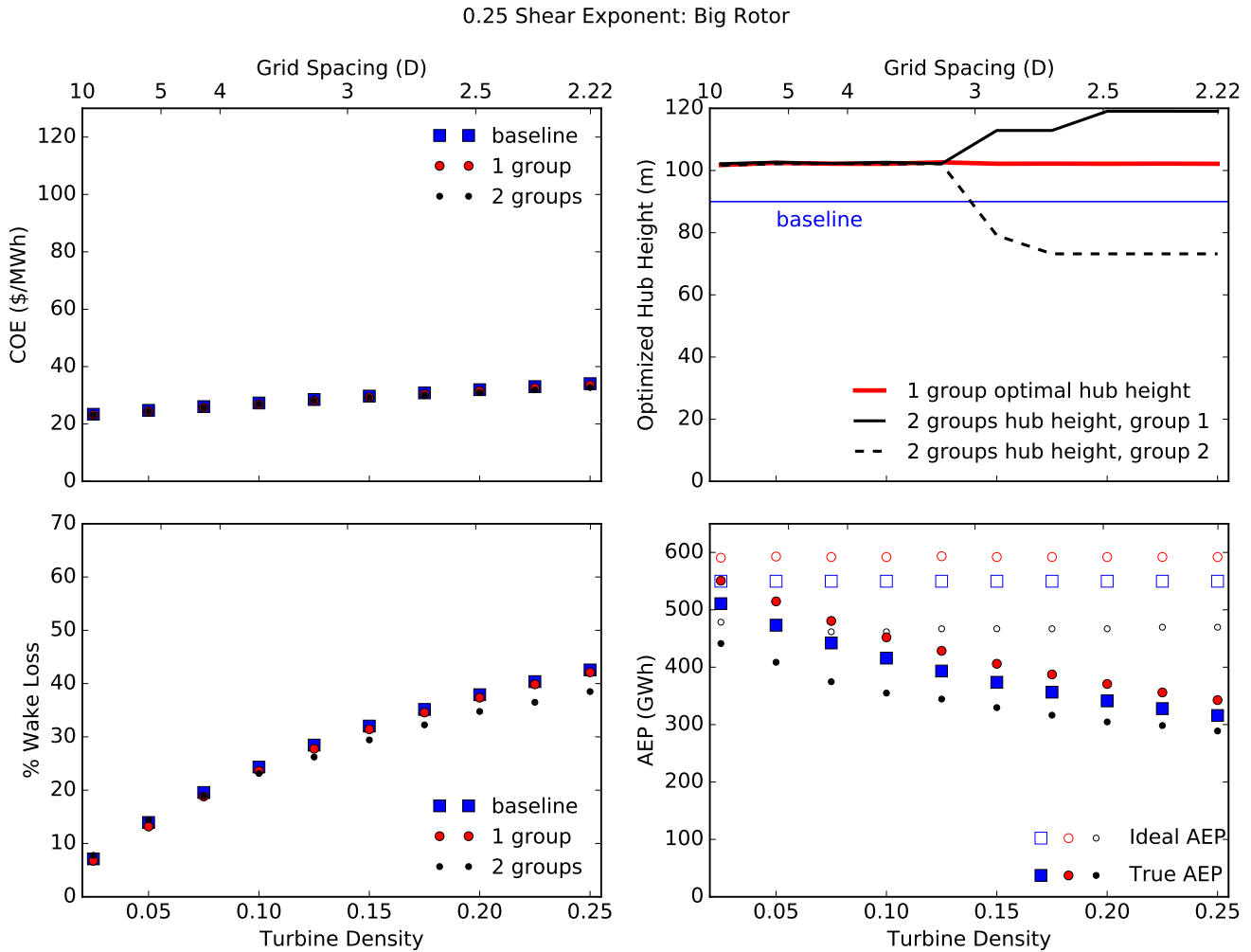


FIGURE 10 Optimization results of a big rotor wind farm while varying the turbine density with a wind shear exponent of 0.25. The top left figure shows the optimized COE as a function of wind shear exponent. The top right figure shows the optimized hub heights of the optimization runs with two height groups. The bottom left figure shows the percent wake loss for the optimized wind farms. The bottom right figure shows the ideal and true AEP for the optimized wind farms.

feasible with gradient-free optimization. The most variables that we optimized in this research were hub height, tower diameter, and tower shell thickness for an 81-turbine wind farm with each turbine varying it individually, a total of 567 design variables.

From our optimization results, we conclude that wind farms with two different hub heights are beneficial in wind farms with small rotor diameters relative to the maximum possible height of the tower, low wind shear, and high turbine density. For a grid wind farm with 81 turbines, 70 meter rotor diameters, a wind shear exponent of 0.08, and 4 diameter spacing between turbines, there is a 4.9% decrease in cost of energy when using two height groups instead of one. For the same farm with 3 diameter spacing between turbines, there is an 11.2% decrease in COE when two height groups are included. A small rotor diameter relative to the maximum hub height attainable allows greater separation between the rotors of different height groups, and minimizes wake interactions between height groups. Although the current trend is towards larger rotor diameters, this finding is still significant because tower heights are also increasing. The absolute rotor diameter is not the important metric, but the relative size of the rotor diameter to the tower height. Low wind shear means similar wind speeds close to the ground as well as higher up, so a shorter height group does not experience significantly lower wind speeds. Farms with high turbine density have tightly packed turbines, meaning there is a large wind velocity deficit in the wakes. In these cases, it is more effective to have different turbine heights to reduce this wake interference.

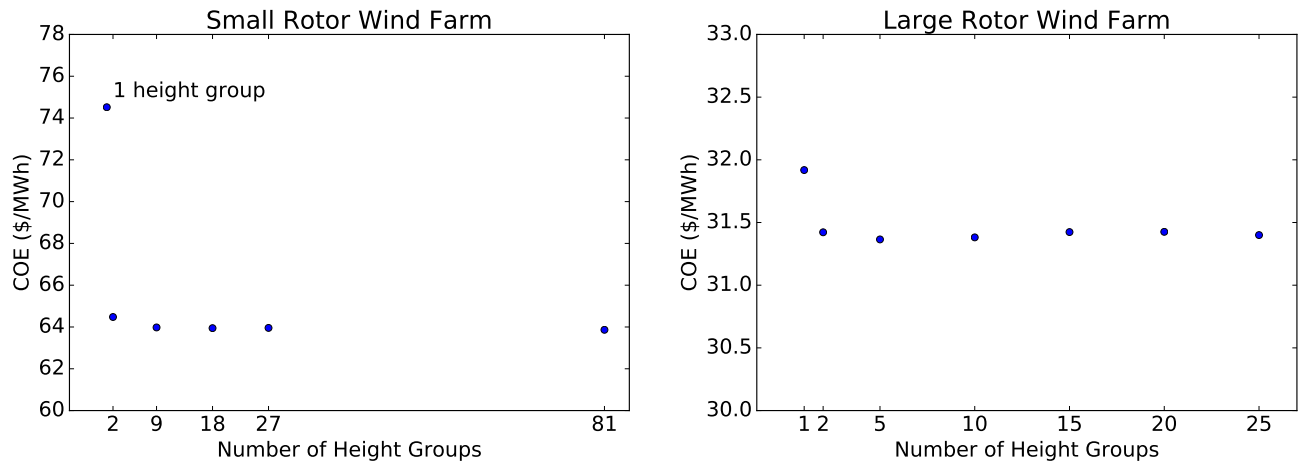


FIGURE 11 The effect of optimized COE as a function of the number of height groups. On the left are results for the small rotor wind farm, on the right are results for the large rotor farm. These are for the small wind farm, and a shear exponent of 0.15.

There can be a large benefit when comparing two height groups to one; however, using more height groups does not provide a significant reduction in COE. The benefit from two height groups to letting every turbine change individually is negligible compared to going from one height group to two. In our opinion, the small benefit is not worth the added complexity.

We suggest several options to continue this research. First is to include other turbine differences beyond hub height. This could include different rotor diameters, turbine ratings, turbine types, blade numbers, cut-in speeds, or several other differences. We expect including other turbine differences in the same wind farm to further decrease wind farm COE. Another continuation of this study will be to further explore the height group turbine assignments and distributions. In this study we assign roughly half of the turbines in the farm to be in each height group. There are likely better ways to assign turbines to a certain height group, and it may be optimal to have more turbines in one height group than the other. Also, because we used basic cylindrical steel tubes, the structural constraints prevented the towers in the large rotor farm to be tall enough for significant height separation between groups. We suggest exploring larger tower diameters and more advanced tower designs to allow the large rotor towers to be much taller, such that different tower heights may be more beneficial. Future work may also benefit from tuning the FLORIS wake model parameters for onshore conditions. In this work, we used parameters that had been tuned for the Princess Amalia wind farm, an offshore farm. Future work should include FLORIS parameters that have been tuned specifically for the wind farms they are modeling. Finally, future work may benefit from using an improved wake model. A recent study by Zhang et al. suggests that wakes close to the ground recover slowly, and as a result, the performance of the shorter turbines in the a multiple-hub-height wind farm can be lower than the performance predicted by an engineering wake model²⁹. Improving the wake model used to consider this phenomenon may be beneficial.

ACKNOWLEDGMENTS

The BYU authors developed this journal article based on funding from the Alliance for Sustainable Energy, LLC, Managing and Operating Contractor for the National Renewable Energy Laboratory for the U.S. Department of Energy.

This work was authored [in part] by the National Renewable Energy Laboratory, operated by Alliance for Sustainable Energy, LLC, for the U.S. Department of Energy (DOE) under Contract No. DE-AC36-08GO28308. Funding provided by the U.S. Department of Energy Office of Energy Efficiency and Renewable Energy Wind Energy Technologies Office. The views expressed in the article do not necessarily represent the views of the DOE or the U.S. Government.

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